Tuning capillary penetration in porous media: Combining geometrical and evaporation effects

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Abstract
Capillary penetration of liquids in porous media is of great importance in many applications and the ability to tune such penetration processes is increasingly sought after. In general, liquid penetration can be retarded or restricted by the evaporation of volatile liquid at the surface of the porous media. Moreover, when capillary penetration occurs in a porous layer with non-uniform cross section, the penetration process can be accelerated or impeded by adjusting the section geometry. In this work, on the basis of Darcy’s Law and mass conservation, a theoretical model of capillary penetration combining evaporation effects in two-dimensional homogeneous porous media of varying cross-section is developed and further examined by numerical simulations. The effects of sample geometry and liquid evaporation on capillary penetration are quantitatively analyzed. Results show that the penetration velocity is sensitive to the geometry of the porous layer, and can be tuned by varying the evaporation rate for a given geometry. Under given evaporation conditions, penetration is restricted to a limited region with a predictable boundary. Furthermore, we find that the inhibition of liquid penetration by evaporation can be offset by varying the geometry of the porous layer. The theoretical model is further extended to model the capillary flow in three-dimensional porous media. The interplay of geometry and evaporation during the capillary flow process in 3D conditions is also investigated. The results obtained can be used to facilitate the design of porous structures to achieve tunable capillary penetration for practical applications in various fields.

1. Introduction

When a dry porous medium contacts a liquid reservoir, the liquid is transported into the porous medium driven by capillary force. This phenomenon is known as capillary penetration. Capillary penetration in porous media is commonly observed in phenomena such as water absorbing into paper [1] and rising damp in concrete walls [2]. Recently, capillary penetration has attracted increasing scientific and industrial attention, owing to the high value of its diverse contemporary applications including paper-based microfluidics [3,4], medical diagnosis [5], energy-harvesting devices [6,7], advanced textile engineering [8,9], cooling devices [10], architectural conservation [11], and oil recovery [12]. Capillary penetration has also been utilized as an inverse method to determine the effective properties (e.g., the pore size distribution and porosity) of porous media in both numerical [13,14] and experimental [15–17] approaches.

Capillary penetration in porous media shares a similar dynamic mechanism with capillary flow in hollow tubes, with both processes resisted by viscous forces [18]. For a capillary tube with an one-dimensional (1D) uniform geometry, dynamic liquid penetration is quantified by a diffusive relationship between the position of the liquid front \( L \) and time \( t \), i.e., \( L^2 = Dt \), where \( D \) is the diffusive coefficient depending on the tube size and the liquid properties [19]. This relationship is best known as the “Lucas–Washburn (LW) law”, presented by Lucas [20] and Washburn [21] a century ago. This classical result has been found to be valid for both unidirectional and radial capillary penetration in porous media [22–25], and is further extended to the hemispherical penetration in a semi-infinite porous medium [26] and imbibition in structured porous media with axially variable geometries [27]. However, recent studies have also indicated that this simple model is not applicable for some complex cases, such as flow processes in...
heterogeneous and random porosity media, and some other effects, e.g., fractal and disorder, should be incorporated into the analysis [28–32].

Tuning capillary penetration processes in porous media, with an emphasis on penetration velocity, is of increasing interest in the burgeoning field of microfluidics [33]. However, the control of fluid flow is not readily achievable by tuning the pore size and porosity for most commonly used porous media such as paper substrates [34]. For the purpose of facilitating control, applied porous media are commonly considered to have different cross sections in the direction of capillary flows [27,35]. Recently, Benner and Petsev [36] pointed out that varying the shape of a porous material leads to qualitative differences in the resulting flow patterns. Additionally, Shou et al. [37] explored the geometry-induced asymmetric capillary flow in porous structures. They found that the geometrical shape has a significant effect on the flow behavior. Subsequently, rational design of porous structures for enhanced and controlled capillary flows have been investigated [38,39]. Notably, previous studies have revealed that the capillary penetration in non-uniform porous structures should be treated as a two- (2D) or three-dimensional (3D) situation, and the time dependence of the flow deviates from the LW 1D case [26,36,39].

A key factor affecting capillary penetration is the evaporation of liquid from the surface of porous media [40,41], with particular relevance for highly volatile liquids, such as the detecting reagent, used in paper-based sensors and diagnostics [42]. Evaporation also has a significant effect on some practical respects, such as soil penetration and water in irrigation processes [43]; liquid flow in fuel cells [44], and liquid transfer in capillary evaporators [45]. Experimental results revealed that capillary penetration in porous media is strongly affected by evaporation, which restricts liquid penetration to a limited region [46]. In order to quantitatively investigate the evaporation effect in unidirectional penetration, the Lucas–Washburn law was employed and augmented by Fries et al. [40] to model dynamic wicking processes. Recently, Liu et al. [47] reported a model based on Darcy’s law and the principle of mass conservation for radial capillary penetration, in which the boundary of the limited liquid penetrated region was predicted theoretically. In their study the geometry was held constant and thus the penetration process was not tunable.

Although capillary penetration in porous systems has been extensively investigated, little work has dealt with the combined effects of geometry and evaporation during capillary penetration. The development of capillary-driven microsystems for energy and biotechnology applications requires precise control and regulation of the penetration process, including velocity and total penetration time. An improved mechanistic understanding of capillary penetration with combined geometrical and evaporation effects will facilitate the application and design of porous structures. From this perspective, we present here a theoretical and numerical study on the combined effects of geometry and evaporation on penetration processes in porous media, with a view towards tunable capillary flow. Porous structures with 2D non-uniform cross-sectional geometrical shape are considered first, and the analysis is further extended to 3D porous structures.

2. Theoretical model

To investigate the combined effects of geometry and evaporation on the capillary penetration through homogenous porous media, a trapezoidal thin porous plate with one end contacting with an unlimited reservoir is considered, as shown in Fig. 1. As a simple case of 2D porous sample with a non-uniform cross section, the trapezoidal structure has essential geometrical features such as asymmetry and shape variation that facilitate the investigation of the basic effect of the sample geometry on fluid penetration in porous media [37]. The penetration processes take place from the liquid reservoir to the other end of the plate (see, Fig. 1a), and is weakened by the concurrent evaporation of liquid from the plate surfaces (see Fig. 1b).

As shown in Fig. 1a, the considered porous plate has a trapezoidal geometrical shape. The width of the plate varies with the position as the following relation,

$$w(\delta) = w_0 \left[1 - \frac{\delta}{l_0}(1 - \alpha)\right],$$

(1)

where $w_0$ is the width of the edge in contact with the reservoir, $l_0$ the length of the plate sample, and $\delta$ the local position in the penetrated region. The thickness of the porous plate $H$ is much smaller than its length dimension $l_0$ such that it can be treated as a planar problem. To study the effect of plate geometry we define a parameter $\alpha = w_1 / w_0$, where $w_1$ is the width of the top edge. The geometry of the plate can also be characterized by the base angle $\beta$ of the trapezoid, which is related to the geometrical factors by

$$\cot \beta = \frac{w_1}{2l_0} \begin{cases} (\alpha - 1), & \alpha \geq 1 \\ (1 - \alpha), & \alpha < 1 \end{cases}$$

(2)

It is noteworthy that the liquid penetration in this trapezoidal porous plate is technically not a simple 1D problem. As discussed in the previous work [36], the liquid flux in the transverse direction of the flow is not generally zero and depends on the opening angle of the expansion (contraction), which is different from the penetration in a uniform plate (i.e., $\alpha = 1$). The liquid front is in general not a straight line perpendicular to the flow direction, which is involved in the 1D case, but an elliptic curve. Therefore, this case should be regarded as a 2D problem. According to the results of Elizalde et al. [39] however, the relative error of flat liquid front assumption in the 1D model, in comparison with the 2D model, is small even if the opening angle as large as 90°, which corresponds to a base angle $\beta$ of 135° in our model. Therefore, to simplify the analysis, we adopt the 1D assumption of flat liquid front (see Fig. 1a) to establish the theoretical model involving the evaporation effect, and the predictions are further examined using numerical simulations in following Section.

We consider the evaporation of liquid from the penetrated region shown in Fig. 1b. A constant and uniform evaporation rate is assumed for a given liquid, ambient temperature, and relative humidity. The total mass flow rate due to evaporation can be calculated as

$$\dot{M}_e = 2 \cdot \dot{m}_e \cdot \frac{l}{2} \left[w(0) + w(l)\right] \cdot l,$$

(3)

where $\dot{m}_e$ is the evaporation rate, which is used to characterize the evaporation of liquid (i.e., the mass loss due to the evaporation per area and time) with the dimension of [kg/m²·s], and $l$ is the length of
the penetrated region. From Eq. (1), w(0) can be calculated as \( w_0 \). Note that the factor "2" in the right side of Eq. (3) indicates the assumption of equal evaporation from front and back surfaces in Fig. 1a. Moreover, evaporation from the secondary side surfaces are neglected owing to the negligible thickness assumed here.

When evaporation is included, mass transfer in porous media conforms to the law of mass conservation. For the considered thin porous plate with porosity \( \phi \) and assuming negligible volume changes, the conservation of mass can be expressed as

\[
\delta \cdot w(\delta) H \phi = l \cdot w(l) H \phi + 2 \frac{m_t}{\rho} \frac{1}{2} [w(\delta) + w(l)] \cdot (l - \delta),
\]

(4)

where \( \rho \) is the fluid density, and \( \delta = d\delta/dt \) and \( l = dl/dt \) are the local fluid velocity and the liquid front velocity, respectively. Furthermore, in the penetration process, the flow rate \( Q \) in a porous medium is obtained based on Darcy’s law as

\[
Q = \frac{\delta A \phi}{\mu} = \frac{Ak}{\mu} \frac{\partial P}{\partial \delta},
\]

(5)

where \( A = w(\delta) \cdot H \) is the cross sectional area of the plate at position \( \delta \), \( k \) is the permeability of the porous medium, \( \mu \) is the viscosity of the liquid, and \( P \) is the pressure of liquid. Combining Eqs. (4) and (5), one obtains

\[
P_c = \frac{\phi |H|}{k} \left[ \frac{l + \frac{b_0}{x-1}}{(x-1)} \ln \left( 1 + \frac{1 + \frac{b_0}{x-1}}{(x-1)} \right) \right]
\]

\[+ \frac{\mu m_t}{kpH} \left[ \left( 1 + \frac{b_0}{x-1} \right)^{\frac{1}{2}} \ln \left( 1 + \frac{1 + \frac{b_0}{x-1}}{(x-1)} \right) - \frac{1}{2} \left( 1 + 2\frac{b_0}{x-1} \right) \right],
\]

(6)

where \( P_c = 2\sigma \cos \theta_0 / R_{ef} \) is the capillary force, governed by the air–liquid surface tension, \( \sigma \), the equilibrium contact angle of the liquid with the solid, \( \theta_0 \), and the effective pore radius of the porous medium, \( R_{ef} \). The second part of the right side in Eq. (6) refers to the evaporation-induced viscous pressure loss \( P_m \), which represents the viscous resistance to liquid front movement in the penetration process. According to the above equation, the penetration distance (i.e., the liquid front position) can be predicted as a function of time. It should be noted that this model is valid only for cases exhibiting a sharp liquid front, i.e., the penetrated region behind the interface is fully-saturated. In addition, its validity is limited to horizontal penetration or vertical penetration without gravity.

Asymmetric penetration is examined by changing the value of \( \alpha \). As a limiting case of uniform cross section, \( \alpha = 1 \), Eq. (6) is reduced to the solution provided by Fries et al. [40]. Another limiting case is that of negligible evaporation, \( m_t = 0 \), whereby Eq. (6) reduces to the classical model [38]. When \( \alpha = 1 \) and \( m_t = 0 \) both meet, the original LW relation [20, 21] is recovered.

3. Numerical simulation and results analysis

3.1. Numerical methods

As mentioned in the preceding section, a flat liquid front is assumed in the theoretical model, which is only accurate and valid for the 1D situation. COMSOL Multiphysics 5.2a finite element software (COMSOL Inc., Burlington, MA) is used to simulate a full 2D problem to examine the proposed model (i.e., Eq. (6)).

The 2D capillary penetration process in porous media can be controlled by a set of simultaneous partial differential equations, viz., Darcy’s law

\[
v = -\frac{k}{\mu} \nabla P,
\]

(7)

and the conservation of mass

\[
F = \rho \left( \frac{k}{\mu} \nabla P \right).
\]

(8)

where \( \rho \) is the fluid density, \( k_i = k/\phi \) is the interstitial permeability, and \( F \) is a source term, which is related to the evaporation of liquid. For negligible evaporation, \( F \) is set to zero; otherwise, it can be calculated as \( F = m_t / H \phi \).

The above Eqs. (7) and (8) govern the 2D penetration process in the full region. When the cross section of the region is uniform, they reduce to the 1D case, as given by Eqs. (5) and (4) with \( \alpha = 1 \). By solving the set of equations, we can obtain the distribution of liquid flow velocity and the boundary of the penetration region at any time.

In finite element analysis, the boundary conditions of the numerical model are defined such that the left and right surfaces are non-penetrable and symmetric and the reservoir-contacting boundary is stationary. Note that the reservoir is exposed to the ambient atmosphere and thus the atmospheric pressure, \( P_{atm} \), is fixed at the reservoir-contacting edge, while the pressure at the fluid front is set as \( P_{atm} = P_c \), where \( P_c \) is the capillary pressure.

To track the velocity of a moving liquid front, a Moving Mesh Module is used in the COMSOL software [35]. To accommodate upward motion of the boundary of the fluid front, side boundaries are allowed to be stretched in the direction of flow, but not in the direction perpendicular to flow. As the liquid front moves automatically, re-meshing is implemented to avoid mesh distortion. Moreover, a mesh sensitivity study has also been conducted a priori to ensure the convergence of the numerical models.

3.2. Results and analysis

By applying the developed numerical model, the capillary penetration process can be simulated for different geometrical shapes and evaporation conditions. The simulation results of the time dependent penetration distance are shown in Fig. 2 as symbols. Also included are the theoretical predictions given by Eq. (6), represented by lines. Two sets of geometrical shapes, (a) \( \alpha = 100 \) (with three base angles \( \beta = 105^{\circ}, 120^{\circ}, \) and \( 135^{\circ} \)) and (b) \( \alpha = 0.01 \) (with \( \beta = 45^{\circ}, 60^{\circ}, \) and \( 75^{\circ} \)) are considered under different evaporation rates (i.e., \( m_t/m_{C}^2 = 0, 1, 2 \) and 5). The results are made dimensionless by two scaling parameters \( b_0 \) and \( t_0 \), where \( b_0 \) is the length of the plate sample, and \( t_0 = \mu |H|/2kP \), is the time scale of the liquid penetrating from the reservoir to the other end of a uniform porous plate (i.e., the special case of \( \alpha = 1 \)). The parameter \( m_{C}^2 \) is the critical evaporation rate that allows liquid to reach the end of the porous plate away from the reservoir. The definition and characteristics of \( m_{C}^2 \) will be discussed later. For illustration, the representative simulated results of penetration region with pressure profiles (i.e., (a) \( \beta = 120^{\circ} \) and (b) \( \beta = 60^{\circ} \) with \( m_t/m_{C}^2 = 0 \) and 2) are included as inserts in Fig. 2.

It can be clearly seen from the inserts in Fig. 2 that the liquid fronts are not straight lines in the simulations. For \( \alpha = 100 \), the liquid front is a convex curve (relative to the reservoir position), but a concave curve for \( \alpha = 0.01 \). For simplification, we choose the middle point of the concave/convex curve to represent the liquid front position, with results are plotted in Fig. 2 as symbols. It should be noted that other selections of the representative point for the liquid front position are also feasible. Although those selections may lead to different relative errors, they do not affect the conclusion drawn here. Numerical simulation and theoretical predictions match closely for all geometrical shapes and evaporation conditions. Specifically, relative error for all three cases of \( \alpha = 100 \) (i.e., \( \beta = 105^{\circ}, 120^{\circ}, \) and \( 135^{\circ} \)) is less than 5%. The small error found may arise owing to the liquid flow velocity, which depends on the pressure gradient following Darcy’s law, and is almost the same at the centre line in 1D and 2D conditions. However, for the set of
Penetration process in porous media is of emerging interest for capillary-driven devices for energy and biotechnology applications. 

4.1. Transition of the penetration velocity

Capillary-driven devices for energy and biotechnology applications require precise control of the capillary penetration velocity. As shown in the preceding section, for some porous samples with special geometrical shapes under appropriate evaporation conditions, the penetration velocity can increase with time following an initial deceleration. This interesting feature of tunable capillary penetration has ramifications towards potential multifunctional applications in microfluidic devices for chemical analysis and catalysis, which necessitate the precise control of flow velocity [48].

Variation of the normalized capillary penetration distance with normalized time for six geometries (\(\alpha = 0, 0.1/\varepsilon, 0.2/\varepsilon, 0.5/\varepsilon, 1/\varepsilon, 3/\varepsilon\) and \(10/\varepsilon\), with \(\varepsilon\) being Euler’s number) are plotted in Fig. 3. As is evident from the inflection of the plotted curves, the penetration velocity transitions from a decreasing to an increasing trend during the penetration process for samples with \(\alpha \leq 1/\varepsilon\). The transition points are marked with diamond symbols for each line (see Fig. 3), and occur at increasing distances for \(\alpha\) values from 0 to 1/\(\varepsilon\). When \(\alpha > 1/\varepsilon\), the transition disappears and the penetration velocity decreases throughout the entire process. The variation of the transition position with the geometrical factor is shown in the insert in Fig. 3.

4. Tuning capillary penetration in porous media

With the sustained development of capillary-based microsystems, such as paper-based microfluidics, medical diagnosis and energy-harvesting devices [3–7], precisely tuning of the capillary penetration process in porous media is of emerging interest for both scientists and engineers in recent years.
The penetration velocity \( v = \frac{dl}{dt} \) can be determined from Eq. (6). Noting that \( dv/dt = 0 \) at the transition position \( l_c \), the relationship between evaporation rate, geometrical factor and transition position can be obtained from Eq. (6), as

\[
\frac{m_c}{m_{eu}} = \frac{2(\alpha - 1)^2\left\{1 + \ln \left[1 + \frac{l_c}{l_0}(\alpha - 1)\right]\right\}}{\left[1 + \left(1 + \frac{l_c}{l_0}(\alpha - 1)\right)^2\right]\ln \left[1 + \frac{l_c}{l_0}(\alpha - 1)\right] - 2\left[1 + \frac{l_c}{l_0}(\alpha - 1)\right]^2\ln \left[1 + \frac{l_c}{l_0}(\alpha - 1)\right] - (\alpha - 1)\frac{2 + \frac{l_c}{l_0}(\alpha - 1)}{1 - \frac{l_c}{l_0}(\alpha - 1)}}.
\]

(9)

where \( m_{eu} = P_c \cdot k \rho H/\mu \) is a scaling parameter used to make the evaporation rate dimensionless. For negligible evaporation, the above equation reduces to an explicit expression for the transition position as

\[
l_c = l_0 \left(1 - e^{-\frac{1}{\alpha - 1}}\right).
\]

(10)

This equation relates the transition point to the geometrical factor as shown in the insert of Fig. 3.

When the evaporation effect is non-negligible, by using Eq. (9), the transition position \( l_c/l_0 \) can be predicted as a function of \( m_e/m_{eu} \) and \( \alpha \) as illustrated in Fig. 4a. In the blue region, the penetration exhibits acceleration, while the decelerating penetration is described by the green region. The interface between these two regions presents the transition position of penetration velocity \( l_c/l_0 \) and its dependence on parameters \( m_e/m_{eu} \) and \( \alpha \), which is predicted quantitatively by Eq. (9). In this 3D phase diagram, the interface between deceleration and acceleration reduces to a linear boundary when evaporation is negligible, which has been discussed before and given in a special case as shown in the insert in Fig. 3.

In order to more clearly show the transition position of penetration velocity for different geometry and evaporation conditions, \( l_c/l_0 \) is plotted as a function of \( m_e/m_{eu} \) for different values of \( \alpha \) in Fig. 4b. It can be found that the transition to accelerating penetration occurs at higher positions for higher evaporation rates for different geometries. Furthermore, for a given evaporation rate, the transition position is higher for larger geometrical factors. Based on such charts, the transition position can be readily determined for analytical purposes, and for informing the design of porous structures, such as chemical detection devices with controllable reaction times applied at different temperatures [5]. It should be mentioned that tuning the penetration velocity, especially the acceleration, necessitates the informed control of sample properties and environmental conditions.

4.2. Evaporation limited capillary penetration

As previously mentioned, the evaporation effect acts as a viscous resistance to the liquid front moving in the penetration process [40,47]. When the evaporation-induced viscous pressure loss equals the capillary pressure (viz., \( P_c = P_m \)), the velocity of liquid front reduces to zero, and the penetration will be restricted to a limited region with a critical length (see Fig. 2). By considering the critical condition of \( P_c = P_m \), the critical length \( l_c \) can be obtained as

\[
\left(\frac{l_c}{l_0} + \frac{1}{\alpha - 1}\right)^2 \ln \left[1 + \frac{l_c}{l_0}(\alpha - 1)\right] - \frac{l_c}{2l_0} \left(\frac{2}{\alpha - 1} + \frac{l_c}{l_0}\right) = \frac{m_c}{m_{eu}}.
\]

(11)

where \( m_{eu} = P_c \cdot k \rho H/\mu \) is the critical evaporation rate for a uniform porous plate that allows penetration through the entire material. This value has been used as a scaling parameter to normalize the evaporation rate in Eq. (9). Furthermore, if we focus on a special circumstance of the critical length exactly equaling to the length of the plate, i.e., \( l_c = l_0 \), the critical evaporation rate \( m_c^e \) can be given as

\[
\frac{m_c^e}{m_{eu}} = \frac{2(1 - \alpha)^2}{1 - \alpha^2 + 2\alpha^2 \ln \alpha}.
\]

(12)

This equation reveals the relation between \( m_c^e \) and \( m_{eu} \). The right hand side will reduce to 1 while \( \alpha = 1 \), meaning \( m_c^e = m_{eu} \) for the case of a uniform porous plate.

According to Eq. (11), the critical length of the evaporation-limited penetration region \( l_c/l_0 \) is plotted as a function of the evaporation rate \( m_e/m_{eu} \) and geometrical factor \( \alpha \) in Fig. 5a. It is clear that when evaporation rate is smaller than a critical value, i.e.,
from Fries et al. [40]. Vargaftik et al. [49] and Rausch et al. [50], water and HEF-7500 under different temperatures are retrieved respectively. Fig. 6 represents the linear relation between critical evaporation rate and the sample size and temperature. These findings demonstrate how liquid penetration can be tuned by changing ambient conditions, adopting different liquids and controlling porous media geometries.

4.3. Geometry-based compensation of penetration time

In the above analyses, we have found that the penetration process is affected not only by liquid evaporation but also by the geometry of the porous media. On one hand, the penetration process will be retarded by evaporation, on the other hand, changing the geometrical shape of the porous sample affects penetration in both accelerating and decelerating regimes. It indicates that the evaporation retarded penetration can be compensated by choosing a proper geometrical shape of the porous sample. In other words, we can tune the capillary penetration in porous media by combining the geometry and evaporation effects. This inspires us to design suitable porous structures for particular using environment.

For capillary penetration in porous media, the total penetration time is an important parameter attracting significant attention in industrial applications such as chemical analysis [37,38]. As shown in Fig. 7a, the normalized critical time for penetration in the whole region (i.e., $t_c/t_0$, where $t_0$ is the critical penetration time of a uniform porous plate without evaporation) is plotted as function of normalized evaporation rate for each geometrical factor, and approaches infinity when the evaporation rate tends to a critical value. After exceeding this critical evaporation rate, the penetration will be restricted to a limited region within the sample. The dotted line in Fig. 7a, which
extends horizontally at $t_c/t_0 = 1$, reveals that tailoring the geometrical shape of a porous sample can compensate for the evaporation-retarded penetration.

To further understand this compensation mechanism, as shown in Fig. 7b, we plot the distribution of penetration time in a phase diagram in the space of $m_e/m_{ew}$ and $\alpha$. It can be seen that there are three regions in two main parts, i.e., the complete penetration part (i.e., $L = L_0$, the liquid can penetrate into the whole sample, including the shaded blue and cyan regions) and the incomplete penetration part (i.e., $L < L_0$, the liquid is limited to a finite region, see the shaded yellow region), with a purple solid line boundary, which indicates the limiting condition of complete penetration with infinite time (i.e., $t_c = \infty$). In the complete penetration region, there is a boundary between the blue and cyan regions shown as a green solid line, which is transplanted from the dotted line in Fig. 7a, and gives the consistence condition for the geometrical shape compensating the evaporation-retarded penetration time to be a constant (i.e., $t_c = t_0$). In the blue region, the total penetration time $t_c$ under the combination of evaporation rate and geometrical factor is smaller than the critical value $t_0$ (i.e., $t_c < t_0$). Thus, this region can be considered as an overcompensated region. In contrast, under-compensation is found in the cyan region, that is, the total penetration time is larger than the critical value, i.e., $t_c > t_0$.

In practical applications, the reference state for compensation may not always be the uniform porous plate. For any given reference state, the consistence condition of $\alpha$ and $m_{ew}/m_{ew}$ can be obtained by moving the dotted line in Fig. 7a up (for the case of $\alpha > 1$) or down (for $\alpha < 1$). Accordingly, the position of the green solid line in the phase diagram in Fig. 7b also needs to be adjusted corresponding to the dotted line. This compensation mechanism can be used as a basis to precisely control penetration processes in chemical analytical devices, with implications towards improving their accuracy [34,48].

5. Compensation mechanism in 3D porous structures

In the previous sections, by applying the proposed theoretical model, the combined effects of geometry and evaporation on the capillary penetration in 2D porous structures are investigated systematically. Nevertheless, capillary flow processes are also commonly found in 3D porous structures, such as capillary evaporators, drug delivery systems, and construction and geotechnical structures [10,11,45], and most of the 3D porous structures have non-uniform cross sections [27]. To fill this gap, we extend the proposed 2D theoretical model to cover 3D porous structures and investigate the interplay of sample geometry and liquid evaporation during the capillary flow process.

To investigate the capillary flow process in 3D structures, we choose a hollow circular frustum cone (see Fig. 8a) and a hollow square frustum pyramid (see Fig. 8b) as two representative 3D porous structures. All the geometrical parameters of these structures are discussed in Appendix A. Two special cases of the geometrical shapes, i.e., uniform thickness and variable thickness, are considered in the following. According to the extended theoretical models, i.e., Eqs. (A.5) and (A.6), the capillary penetration process in 3D porous structures can be predicted for samples with different geometrical shapes and under different evaporation conditions, see Appendix A for the details.

As we have discussed in Section 4.3, the evaporation-retarded penetration can be compensated by tailoring geometrical shape of the porous sample. For the 2D porous sample, only the geometric factor is tunable, but for the 3D porous samples, one more parameter (i.e., the relative layer thickness $P$, its definition given in Appendix A) can be introduced to increase the design flexibility. Here we also choose the critical penetration time for the whole region ($t_c$) as the object variable of the compensation, and the corresponding reference is the critical penetration time of a uniform porous cylinder without evaporation ($t_0$).

First, we focus on the 3D hollow porous structure with uniform thickness. As shown in Fig. 9, the normalized critical penetration time (i.e., $t_c/t_0$) is plotted as a function of evaporation rate for porous samples with different geometrical shapes (i.e., $\alpha = 0, 0.1, 0.25, 0.5, 1.0$ and 2.0). For each geometry, three samples with different relative thicknesses, i.e., $P = 0.01 \alpha, 0.5 \alpha$ and $\alpha$, are considered. When the relative layer thickness approaches zero (e.g., $P = 0.01 \alpha$), the results of 3D cases are consistent with the 2D results (as shown by symbols). Similar to the 2D cases (see Fig. 7(a)), the penetration time increases with increasing evaporation rate for each sample with different geometric factors and relative thicknesses. Additionally, the penetration time approaches infinity when the evaporation rate tends to a critical value, and the limited penetration occurs when the evaporation rate exceeds the critical value.

For the samples with a given geometric factor, the relation between critical penetration time and evaporation rate is dependent upon the relative layer thickness of the sample. However, it can be interestingly found that this dependence vanishes at the state of $t_c/t_0 = 1$, which reveals that the consistence condition of compensation for the evaporation-retarded penetration does not depend on the relative layer thickness. By extracting the
When the evaporation rate is larger enough, the liquid penetration is limited in a finite region. By employing the equilibrium condition of $P_e = P_m$, the critical condition for the complete penetration (i.e., the liquid can penetrate into the whole sample exactly) can be obtained as

$$\frac{m_{\text{eu}}}{m_{\text{eu}}^{\text{crit}}} = \frac{\alpha^2 - \Pi^2/4}{(1 - \alpha)^2} \ln \frac{\alpha - \Pi/2}{1 - \Pi/2},$$

$$\frac{1 + \Pi/2}{1 - \alpha}.$$

Eq. (13) reduces to Eq. (12), which is governing the critical condition of the geometrical factor in Fig. 10(b). Five cases with different relative thicknesses, i.e., $\Pi = 0.01, 0.25, 0.50, 0.75$ and 1.00, are considered here, and the case with $\Pi = 0.01$ is consistence with the 2D results (as shown by symbols). Due to the relative thickness confined by $\Pi \leq \alpha$, the critical condition between the geometrical factor and evaporation rate is truncated at $\alpha = \Pi$ for each case. Additionally, for a given geometrical factor, larger relative thickness corresponds to smaller critical evaporation rate, meaning the complete penetration is easier to attain for the sample with larger relative thickness.

All the discussions presented above are focusing on the 3D hollow porous structures with uniform layer thickness. However, we cannot access the properties of solid cylindrical porous sample by changing the geometrical parameters, even if the inner radial reduces to zero. For this reason, we consider the liquid penetration process in 3D hollow porous structures with variable layer thickness. With the proposed model, i.e., Eq. (A.6), the variation
of the liquid front with penetration time can be calculated for 3D hollow porous structures with different geometrical shapes under different evaporation conditions.

When the evaporation is negligible, i.e., \( \dot{m}_e = 0 \), by integrating Eq. \((A.6)\), the relation between penetration distance and penetration time can be obtained explicitly as \( h^2 - 2/3 \cdot (1 - x)h^3 = t \) with \( t = t/t_0 \). For the complete penetration, i.e., \( h = 1 \), the critical condition can be further simplified as \( 1/3 + 2x/3 = t_c \), where \( t_c = t_c/t_0 \) is the normalized critical penetration time. The critical penetration time \( t_c \) is plotted as a function of the geometrical factor \( x \) in Fig. 11(a) with semi-log coordinates. The asymmetric capillary flow is clearly found when \( x \) varying from 0.1 to 10. Moreover, the linear variation of \( t_c \) with respect to \( x \) is also shown in the insert of Fig. 11(a) with initial value of \( t_c = 1/3 \) when \( x = 0 \).

When the evaporation is non-negligible, as we have discussed before, the evaporation-retarded penetration can be compensated by changing the geometrical shape of the porous sample. For the 3D hollow porous structures with variable layer thickness, the normalized critical penetration time can be plotted as a function of evaporation rate for samples with different geometrical shapes (i.e., \( x = 0.5, 1.0 \) and \( 2.0 \)), as shown in Fig. 11(b). Three samples with different relative thicknesses, i.e., \( \Pi = 0.01, 0.5 \) and \( \Pi \), are considered for each geometry. It is clear that the relative thickness has no effect on the critical time when the evaporation is negligible (i.e., \( \dot{m}_e = 0 \)). With the evaporation rate increasing, the significance of the effect of relative thickness on the critical time increases. Following the horizontal dotted line in Fig. 11(b), one can find that the consistence condition of compensation is dependent on the relative layer thickness, which is different from the results of 3D structures with uniform layer thickness. Additionally, the critical condition of complete penetration also depends on the relative thickness.

The findings presented in this section provide us clearer understanding on the capillary flow in 3D porous structures, especially the compensation mechanism. By means of these findings, the design of capillary flow devices can be improved to facilitate the applications in a wide range, especially for the devices with high precision and sensitivity requirements.

6. Conclusion

In summary, a general framework has been developed to quantitatively investigate the significant effects of sample geometry and the liquid evaporation on capillary penetration processes in both 2D and 3D porous media with non-uniform cross sections. By combining the effects of geometry and evaporation, the velocity and extent of capillary penetration can be tuned with implications for diverse practical applications. Numerical simulations have been performed to support the developed theoretical model for 2D porous media. Good agreement is obtained for the base angles between 45° and 135°.

For 2D porous media, the velocity of capillary penetration is found to be sensitive to the geometry of the porous sample, and it is found to transition from a decreasing to an increasing trend during the penetration process for a given geometry. Moreover, the distribution of penetration velocity can be modified by evaporation effects. The transition position has been predicted quantitatively as a function of the evaporation rate and the geometrical factor. Furthermore, a critical value of the evaporation was found to exist, above which liquid penetration is restricted to a limited region with a predictable boundary. The critical length of the limited penetration region depends on the evaporation rate and geometrical factor. It is particularly interesting to note that evaporation-retarded penetration can be compensated by choosing an appropriate sample geometry, and the consistence condition is given theoretically.

The capillary penetration in 3D porous structures was investigated. It is found that the relative layer thickness has a significant effect on the penetration process and the critical condition of the complete penetration in 3D structures for both cases with uniform and variable thickness. Another interesting finding is that the compensation condition for the evaporation-retarded penetration in 3D porous structures with uniform thickness does not depend on the layer thickness, but such dependence is found for porous structures with variable thickness.

The present analysis provides a useful framework to investigate the underlying mechanisms of penetration processes by combining geometry and evaporation effects. Insights gained from this work warrant new designs of more complex and actual porous architectures to achieve active control of the capillary penetration processes for a wide range of practical applications. Furthermore, it is worth mentioning that the present framework to consider geometrical factors can be applied to alternative shapes, other than the trapezoid and hollow circular/square frustum discussed here.

Competing financial interests

The authors declare no competing financial interests.

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Appendix A. Theoretical model for capillary flow in 3D structures

For the 3D porous structures shown in Fig. 8, when the porous samples attach with liquid reservoir, the liquid will transfer to the other end from the reservoir driven by capillary pressure. For the hollow circular frustum (see Fig. 8a), it is clear that the structure is axisymmetric. Its intermediate cross-section, i.e., the plane of symmetry, is an isosceles trapezoid. According to the symmetry, half of the intermediate cross-section, i.e., \( O_1O_2Q_1Q_2 \), can be illustrated in Fig. 8c. For the hollow square frustum (see Fig. 8b), the axisymmetry is not valid, but half of symmetrical plane \( O_1O_2Q_1Q_2 \) can also be chose to characterize the basic geometric features of the whole structure. Similar to the analysis of 2D cases in Section 2, half of the width and the porous layer thickness are assumed to vary with the position as

\[
R(z) = R_0 \left( 1 - \frac{z}{h_0} (1 - x_0) \right), \quad (A.1)
\]

and

\[
T(z) = T_0 \left( 1 - \frac{z}{h_0} (1 - x_0) \right), \quad (A.2)
\]

where \( R_0 \) and \( T_0 \) are the half width and layer thickness at the position in contact with the reservoir, respectively, \( h_0 \) is the length of the central axis of the hollow circular frustum cone and the hollow square frustum pyramid, \( z \) is the local position in the penetrated region along the central axis, and \( x_0 \) and \( x_r \) are the geometric parameters with respect to the half width and layer thickness, respectively. Similar to the 2D trapezoidal porous plate, the geometric feature of these 3D porous sample can be characterized by the geometric parameters \( x_0 \) and \( x_r \).

Similar to the analysis of 2D trapezoidal structures, here we adopt the 1D assumption of flat liquid front (see Fig. 8c) in the theoretical model to investigate the basic characteristics of capillary flow in the 3D structures. In addition, when the liquid penetrates through these hollow structures, evaporation may take place at both the inner and outer surfaces of the penetrated region. To simplify, only the evaporation at the outer surface is considered, as shown in Fig. 8c.

For the hollow circular frustum (see Fig. 8(a)), the mass conservation under the evaporation condition can be expressed as

\[
\dot{z} \cdot \pi [R^2(z) - (R(z) - T(z))^2] \phi = \dot{h} \cdot [R^2(h) - (R(h) - T(h))^2] \phi - \frac{m_e}{\rho} \cdot \pi \sqrt{(h - z)^2 + [R(z) - R(h)]^2} \cdot [R(z) + R(h)], \quad (A.3)
\]

where \( z \) and \( h \) indicate the local position in the penetrated region and the liquid front position, respectively, as shown in Fig. 8(c).

Technically, one can obtain the fully control equation for the moving liquid front position by combining Eqs. (5) and (A.3). However, it should be noted that the pressure gradient, which is used to determine the flow rate in Eq. (5), is depending on the flow direction. Here, we use the inner boundary of the porous layer, i.e., \( \tau_{T'_{rT}} \) in Fig. 8c, as the flow direction. Introducing the slant angle \( \gamma \), Eq. (5) can be rewritten, and the control equation can be obtained by combining with Eq. (A.3) as

\[
P_c' = \frac{\phi \mu}{k} \int_0^h 2dz = \frac{\phi \mu}{k} \int_0^h \frac{2R(h)T(h) - T(h)^2}{2R(z)T(z) - T(z)^2} \cdot \frac{h}{R(z) + R(h)} \cdot \sqrt{\left( \frac{h}{R(z) - R(h)} \right)^2 + [R(z) - R(h)]^2} \cdot \frac{h}{R(z) + R(h)} \cdot T(z) - T(z)^2} dz.
\]

(A.4)

where \( P_c' = P_c \cdot (\sin \gamma)^2 \), in which \( P_c \) is the capillary force with the same expression as 2D case as \( P_c = 2 \pi \sigma \cos \phi / \gamma_k \).

By substituting the expressions of \( R(z) \), \( T(z) \), \( R(h) \) and \( T(h) \), through Eqs. (A.1) and (A.2), into Eq. (A.4), the penetration distance, i.e., the liquid front position, can be calculated as a function of penetration time. It should be noted that the mass conservation of the hollow square frustum (see Fig. 8(b)) has a similar expression to Eq. (A.3) for the hollow circular frustum. We can further confirm that Eq. (A.4) is also valid for the hollow square frustum.

Noted that there are two integral functions in Eq. (A.4), its explicit expression is not easy to obtain for general cases. Here we focus on two special cases, i.e., uniform thickness and variable thickness, to derive explicit expressions of the control equation. When the 3D hollow porous structure has a uniform thickness, i.e., \( T(z) = T_0 \), the geometrical parameters can be set to \( x_0 = x \) and \( x_r = 1 \). Following these conditions, the control equation can be obtained as

\[
P_c = \frac{\phi \mu t_k}{k} \frac{1 - \pi/2 - (1 - x)h}{1 - x} \ln \frac{1 - \pi/2 - (1 - x)h}{1 - (1 - x)h} \frac{dh}{dt} + \frac{\frac{m_e}{\rho}}{2pkr_0} \left\{ \frac{1 - (1 - x)h^2}{(1 - x)^2} - \pi^2/4 \right\} \ln \frac{1 - (1 - x)h}{1 - \pi/2} \frac{dh}{dt} = \frac{h^2}{2} \frac{(1 + \pi/2)h}{1 - x}.
\]

(A.5)

where \( l_0 \) is the distance of the sample along the flow direction and can be related to the axial length \( h_0 \) via \( h_0/l_0 = \sin \gamma \). Here, \( h = h_0 \) is the projected length of the penetrated region, \( T_0 = T_0/\sqrt{1 + (1 - z)^2/H^2} \) is the effective thickness of the porous layer at the bottom surface, and \( \Pi = T_0/R_0 \) is the relative thickness with respect to the half width. Following the flat front assumption, one can find that \( l = l_0/l_0 = h \), where \( l \) is the length of the penetrated region along the flow direction.

When the thickness of the porous layer is much smaller than the width of the sample, i.e., \( \Pi \rightarrow 0 \), hollow square frustum pyramid can be unfolded as four trapezoids. Consequently, Eq. (A.5) reduces to Eq. (6) by replacing \( h \) by \( l \). Furthermore, hollow circular frustum cone can be unfolded as a sector ring when \( \Pi \rightarrow 0 \). Hence Eq. (A.5) can reduce to the theoretical model for radial penetration in our previous work [47], by replacing \( h \) by \( r \) through the relation \( r = 1 - h - (1 - x) \).

For the special case of variable thickness, we choose the most simple case, i.e., \( x_0 = x = x_r \), the control equation can be derived as

\[
P_c = \frac{\phi \mu t_k}{k} \left\{ 1 - (1 - x)h \right\} \frac{dh}{dt} = \frac{\frac{m_e}{\rho}}{2pkr_0} \left\{ \frac{1 - (1 - x)h^2}{1 - \pi/2} \right\} \frac{h^2}{2}.
\]

(A.6)

For this case, when \( \Pi = 1 \), the hollow structure becomes solid. Furthermore, when \( x = 1 \), Eq. (A.6) reduces to the classical 1D model given by Fries et al. [40].

Here we focus on the hollow porous sample with uniform layer thickness as an example. As shown in Fig. A.1, the normalized penetration distances are plotted as a function of normalized penetration time for two sets of geometric parameters, i.e., \( x = 2.0 \) and \( x = 0.5 \). For each geometry, three samples with different relative
layer thicknesses, i.e., \( \Pi = 0.01 \alpha, 0.5 \alpha \) and \( \bar{\alpha} \) (where \( \bar{\alpha} = \alpha \) when \( \alpha < 1 \) and \( \alpha = 1 \) when \( \alpha > 1 \)), are considered, as shown by solid lines with different colors. For the purpose of comparison, the prediction of 2D model (Eq. (6)) are also plotted by symbols and definitely consist with the 3D case of \( \Pi = 0.01 \alpha \).

It can be seen from Fig. A.1 that, for both two sets of geometries, the evaporation of liquid weakens the penetration process generally, i.e., higher evaporation rate leads to slower flow velocity for all the three samples with different relative thicknesses. However, for a given evaporation condition, the effect of relative layer thickness on the penetration depends on the geometric shape of the sample. Specifically, for the case of \( \alpha = 2.0 \) under two evaporation rates (i.e., \( \dot{m}_l/\dot{m}_f^o = 0 \) and 1.0), as shown in Fig. A1(a), slower flow velocity can be found for the sample with larger relative thickness. However, for the case of \( \alpha = 0.5 \), it is interesting to find that there is a transition of the effect of relative layer thickness on the penetration from negative to positive by varying the evaporation rates, see Fig. A1(b). When \( \dot{m}_l/\dot{m}_f^o \approx 0 \), slower flow velocities are found for the thinner samples. On the contrary, faster flow velocities are obtained in the thinner samples when \( \dot{m}_l/\dot{m}_f^o \approx 1 \). The transition approximately occurs at \( \dot{m}_l/\dot{m}_f^o \approx 0.3 \), and the flow velocity is independent on the relative layer thickness under this special evaporation condition. This interesting transition phenomenon has significant effects on the compensation mechanism in 3D structures.

References